

DRYER

Technical Field

The present invention relates to a drying machine,
5 and more particularly, to a bearing structure rotatably
supporting a drum of a drying machine.

Background Art

Generally, a drying machine is an apparatus for
drying laundry using hot air generated by a heater. The
10 drying machine comprises a drum for containing laundry, and
the hot air is supplied to the drum. In order to promote
the drying of the laundry, the drum is rotated preferably
by a driving device, and a bearing structure for supporting
a shaft of the rotating drum is provided to the drying
15 machine.

The above conventional bearing structure of the
drying machine has problems, as follows.

First, grease serving as a lubricant leaks from a
bearing while the drying machine is being used, thereby
20 hindering lubricating movements of the bearing and the
shaft.

Second, the bearing is generally made of a metal.
Such a metal bearing cannot be finely manufactured, thereby
causing abnormally excessive abrasion of a journal of the

shaft and noise.

Disclosure of Invention

An object of the present invention devised to solve the problem lies on provision of a bearing structure for
5 stably supporting a rotary drum.

The object of the present invention can be achieved by providing a drying machine comprising: a cabinet; a drum rotatably installed in the cabinet for containing laundry to be dried; a first housing connected to the drum; a
10 second housing connected to the first housing; first and second bearings respectively provided in the first and second housings; and a shaft supported by the first and second bearings, and rotatably connected to the drum.

The first and second bearings may be connected to each
15 other, and preferably may be connected to each other such that they are respectively sealed. In order to form the sealed structure, one of the first and second bearings may comprise ribs formed therein along its edge, and the other one of the first and second bearings may comprise grooves
20 engaged with the ribs.

The shaft may be fixed to the cabinet. Thereby, the first and second bearings are rotated together with the rotation of the drum.

The first and second bearings support the shaft in the

radial direction and the axial direction of the shaft. Thus, the first and second bearings surround the end of a journal of the shaft.

The journal of the shaft may have a spherical shape, and each of the first and second bearings may comprise a recess for mounting the spherical journal.

The journal of the shaft may be made of a metal, and a space for storing a grease may be formed between a journal of the shaft and the first and second bearings. Further, the journal of the shaft may comprise at least one oil groove formed therein along its outer circumferential surface.

The first and second bearing may be formed by injection molding and be made of plastic.

Preferably, the first and second bearings are respectively formed integrally with the first and second housings.

In another aspect of the present invention, provided herein is a drying machine comprising: a cabinet; a drum rotatably installed in the cabinet for containing laundry to be dried; a first bearing connected to the drum; a second bearing connected to the first bearing; and a shaft supported by the first and second bearings, and rotatably connected to the drum.

In a further aspect of the present invention, provided herein is a drying machine comprising: a cabinet; a drum rotatably installed in the cabinet for containing laundry to be dried; a shaft connected to the drum; a housing unit provided on the cabinet, and surrounding the shaft; and a bearing installed in the housing unit, and rotatably supporting the shaft.

Advantageous Effects

In the drying machine, the drum during rotation is stably supported.

Brief Description of Drawings

The accompanying drawings, which are included to provide a further understanding of the invention, illustrate embodiments of the invention and together with the description serve to explain the principle of the invention.

In the drawings:

FIG. 1 is a schematic transversal sectional view of a drying machine in accordance with the present invention;

FIG. 2 is a sectional view of a first embodiment of a bearing structure of the drying machine in accordance with the present invention;

FIG. 3 is an exploded perspective view of the first embodiment of the bearing structure;

FIG. 4 is a perspective view of a first housing observed in the direction of "B" of FIG. 3;

FIG. 5 is a perspective view of a first bearing observed in the direction of "B" of FIG. 3;

5 FIG. 6 is a perspective view of the first bearing observed in the direction of "A" of FIG. 3;

FIG. 7 is a perspective view of a second bearing observed in the direction of "B" of FIG. 3;

10 FIG. 8 is a perspective view of the second bearing observed in the direction of "A" of FIG. 3;

FIG. 9 is a perspective view of a second housing observed in the direction of "A" of FIG. 3;

FIG. 10 is a perspective view of the second housing observed in the direction of "B" of FIG. 3;

15 FIG. 11 is a perspective view of a shaft of the first embodiment of the bearing structure;

FIG. 12 is a sectional view of a modification of the first embodiment of the bearing structure;

20 FIG. 13 is an exploded perspective view of the modification of FIG. 12;

FIG. 14 is a perspective view of a first bearing observed in the direction of "B" of FIG. 13;

FIG. 15 is a perspective view of the first bearing observed in the direction of "A" of FIG. 13;

FIG. 16 is a perspective view of a second bearing observed in the direction of "A" of FIG. 13;

FIG. 17 is a perspective view of the second bearing observed in the direction of "B" of FIG. 13;

5 FIG. 18 is a sectional view of a second embodiment of the bearing structure of the drying machine in accordance with the present invention; and

FIG. 19 is an exploded perspective view of the second embodiment of the bearing structure.

10 Best Mode for Carrying Out the Invention

Reference will now be made in detail to the preferred embodiments of the present invention, examples of which are illustrated in the accompanying drawings. In the following description of the present invention, the same or similar
15 elements are denoted by the same reference numerals and terms and a detailed description thereof will thus be omitted because it is considered to be unnecessary.

FIG. 1 is a schematic transversal sectional view of a drying machine in accordance with the present invention;

20 The drying machine of the present invention comprises a cabinet 1 and a drum 30 rotatably installed in the cabinet 1. An opening 5 is formed through the front surface of the cabinet 1, and laundry is put into the drum 30 through the opening 5. A rib 2, which is formed along

the opening 5, is inserted into the drum 30 and rotatably supports the front portion of the drum 30. The drum 30 is rotated by designated driving devices, for example, a motor and a belt for connecting the motor and the drum 30. Other various mechanisms may be used as the driving device of the drum 30. The drum 30 is rotatably installed on a rear surface 3 of the cabinet 1. A bearing structure is applied between the drum 30 and the rear surface 3 of the cabinet 1 so as to stably support the rotating drum 30. Hereinafter, the bearing structure will be described in detail with reference to corresponding drawings, as below.

FIG. 2 is a sectional view of a first embodiment of a bearing structure of the drying machine in accordance with the present invention, and FIG. 3 is an exploded perspective view of the first embodiment of the bearing structure. As shown in FIGS. 2 and 3, the first embodiment of the bearing structure comprises first and second housings 100 and 200, first and second bearings 150 and 250, and a shaft 50.

The first housing 100 is connected to the drum 30 (FIG. 1), and the second housing 200 is connected to the first housing 100. The first and second bearings 150 and 250 are installed between the first housing 100 and the second housing, and the shaft 50, more precisely, a journal

52 of the shaft 50, is mounted between the first bearing 150 and the second bearing 250.

With reference to FIG. 4, the first housing 100 has an approximately circular shape, and a seat 118 for mounting the first bearing 150 is formed in the central portion of the first housing 100. A plurality of connection holes 134 are formed through the first housing 100 so that the first housing 100 is connected to the drum 30 by connection members (not shown), and a plurality of recesses 105 are formed along the circumference of the seat 118.

With reference to FIGS. 5 and 6, the first bearing 150 has an approximately circular shape, and comprises a recess 178 formed therein for mounting the journal 52 of the shaft 50. Preferably, the recess 178 has a size corresponding to the size of the seat 118 of the first housing 100. As shown in FIG. 6 in detail, the first bearing 150 further comprises protrusions 155 corresponding to the recesses 105 of the first housing 100. The protrusions 155 are inserted into the recesses 105, thereby causing the first bearing 150 to be directly and precisely connected to the first housing 100 without using connection members. The protrusions 155 may be formed on the first housing 100, and the recesses 105 may be formed in the

first bearing 150. In this case, the same function can be achieved. At least one recess 170 is formed along the circumference of the first bearing 150. Grooves 191 and 192 are formed in the surface of the first bearing 150 facing the second bearing 250 along the circumference of the recess 178.

With reference to FIGS. 7 and 8, the second bearing 250 has an approximately circular shape, and comprises a recess 278 for mounting the journal 52 of the shaft 50 and a through hole 277 for passing the shaft 50. The second bearing 250 further comprises at least one rib 270 having a position and size corresponding to those of the recess 170 of the first bearing 150. Accordingly, the second bearing 250 is directly connected to the first bearing 150 by inserting the rib 270 into the recess 170 preferably without using connection members. The rib 270 may be formed on the first bearing 150, and the recess 170 may be formed in the second bearing 250. In this case, the same function can be achieved.

Further, circular ribs 291 and 292 having positions and sizes corresponding to those of the grooves 191 and 192 of the first bearing 150 are formed on the second bearing 250. When the first and second bearings 150 and 250 are connected to each other, the ribs 291 and 292 and the

grooves 191 and 192 are engaged with each other. Accordingly, the first and second bearings 150 and 250, more precisely, the recesses 178 and 278, are tightly sealed by the ribs 291 and 292 and the grooves 191 and 192.

5 The ribs 291 and 292 and the grooves 191 and 192 prevent a grease from leaking from the first and second bearings 150 and 250, and more precisely, from the recesses 178 and 278, during the rotation of the drum. The ribs 291 and 292 may be formed on the first bearing 150, and the grooves 191 and
10 192 may be formed in the second bearing 250. In this case, the same function can be achieved.

With reference to FIGS. 9 and 10, a plurality of connection holes 234 are formed through the second housing 200 at positions corresponding to the connection holes 134
15 of the first housing 100. Thus, the first and second housings 100 and 200 are connected to the drum 30 using the connection holes 134 and 234 and connection members (for example, screws) passing through the connection holes 134 and 234.

20 The second housing 200 comprises a seat 218 for mounting the second bearing 250. Further, the second housing 200 comprises a through hole 287 for passing the shaft 50.

With reference to FIG. 11, the shaft 50 comprises a

body 54, and the journal 52 supported by the first and second bearings 150 and 250. As shown in FIG. 1, the body 54 is fixed to the rear surface 3 of the cabinet 1 using a screw portion 56 formed on the body 54 and a connection member, and the journal 52 is supported by the first and second bearings 150 and 250. The drum 30 is rotatably connected to the shaft 50. Accordingly, while the drying machine is being driven, the shaft 50 is stationary and the drum 30 is rotated against the shaft 50 together with the rotation of the first and second bearings 150 and 250. That is, the shaft 50 rotatably supports the drum 30.

In order to stably support the drum 30, the journal 52 of the shaft 50 substantially has a spherical shape. The first and second bearings 150 and 250 surround the end of the journal 52 so as to stably support the journal 52. More particularly, the recesses 178 and 278 of the first and second bearings 150 and 250 substantially have a hemispherical shape. Thereby, the first and second bearings 150 and 250 support the shaft 50 in the axial direction of the shaft 50 as well as in the radial direction of the shaft 50. As a result, the above bearing structure stably supports the drum 30 as well as the shaft 50.

Preferably, the journal 52 of the shaft 50 has a

cutting portion 58. A designated space 175, as shown in FIG. 2, is formed between the cutting portion 58 of the journal 52 and the first and second bearings 150, and the grease is stored in the space 175. Thereby, the grease is sufficiently supplied to the first and second bearings 150 and 250 for providing effective lubrication. Preferably, at least one oil groove (not shown) is formed in the outer circumferential surface of the journal 52. The oil groove always stores a designated amount of the grease, and supplies the grease to the space 175 between the journal 52 and the first and second bearings 150 and 250 for providing effective lubrication.

The journal 52 is made of a metal, and is formed on the body 54 by injection molding. Since the journal 52 is made of a metal, the abrasion of the journal 52 is drastically reduced. The first and second bearings 150 and 250 are made of plastic. Preferably, the first and second bearings 150 and 250 are also formed by injection molding. Since the first and second bearings 150 and 250 can be finely manufactured and easily finished differing from a bearing made of a metal, surfaces of the first and second bearings 150 and 250 contacting the journal 52 have high surface roughness. Thereby, the abrasion of the journal 52 is more reduced, and the generation of noise is reduced

also.

FIGS. 12 to 17 illustrate a modification of the first embodiment of the bearing structure. In the above modification, first and second bearings are substantially
5 formed as a single body with first and second housings. Hereinafter, the modification of the first embodiment will be described in more detail.

FIG. 12 is a sectional view of the modification of the first embodiment of the bearing structure, and FIG. 13
10 is an exploded perspective view of the modification of FIG. 12. As shown in FIGS. 12 and 13, the modification comprises first and second bearings 300 and 400, and a shaft 50.

The first bearing 300 is connected to the drum 30
15 (FIG. 1), and the second bearing 400 is connected to the first bearing 300. The shaft 50, and more precisely, the journal 52 of the shaft 50, is disposed between the first bearing 300 and the second bearing 400.

With reference to FIGS. 14 and 15, the first bearing
20 300 comprises a boss 350 protruded from the central portion thereof, and a recess 358 formed in the boss 350 for mounting the journal 52. The first bearing 300 further comprises a plurality of connection holes 333 formed along the edge of the first bearing 300 so that the first bearing

300 is connected to the drum 30 by the connection holes 333. Preferably, as shown in FIG. 15, in order to reinforce the strength of the first bearing 300 and connection portions, the connection holes 333 are respectively formed in
5 corresponding bosses 338. A circular rib 310 and a circular groove 345 are formed on and in the first bearing 300 along the circumference of the recess 358.

With reference to FIGS. 16 and 17, the second bearing 400 comprises a seat 450 for mounting the boss 350 of the
10 first bearing 300, and a recess 458 formed in the seat 450 for mounting the journal 52 of the shaft 50. A through hole 450 for passing the shaft 50 is formed through the recess 458. A plurality of connection holes 433 formed through the second bearing 400 along its edge at positions
15 corresponding to the connection holes 333 of the first bearing 300. Preferably, as shown in FIG. 17, in order to reinforce the strength of the second bearing 400, the connection holes 433 are respectively formed in corresponding bosses 438. A circular rib 410 and a
20 circular groove 445 are formed on and in the second bearing 400 along the circumference of the recess 458 so that the circular rib 410 and the circular groove 445 correspond to the circular groove 345 and the circular rib 310 of the first bearing 300. When the first and second bearings 300

and 400 are connected to each other, the circular rib 310 and the circular groove 345 are engaged with the circular groove 445 and the circular rib 410. Accordingly, the first and second bearings 300 and 400, more precisely, the recesses 358 and 458, are tightly sealed by the circular ribs 310 and 410 and the circular grooves 345 and 445. The circular ribs 310 and 410 and the circular grooves 345 and 445 prevent a grease from leaking from the first and second bearings 300 and 400, and more precisely, from the recesses 358 and 458, during the rotation of the drum.

The shaft 50 comprises a body 54, and the journal 52 supported by the first and second bearings 300 and 400. As shown in FIG. 1, the body 54 is fixed to the rear surface 3 of the cabinet 1 using a screw portion 56 formed on the body 54 and a connection member, and the journal 52 is supported by the first and second bearings 300 and 400. The drum 30 is rotatably connected to the shaft 50. Accordingly, while the drying machine is being driven, the shaft 50 is stationary, and the drum 30 is rotated against the shaft 50 together with the rotation of the first and second bearings 300 and 400. That is, the shaft 50 rotatably supports the drum 30.

In order to stably support the drum 30, the journal 52 of the shaft 50 substantially has a spherical shape.

The first and second bearings 300 and 400 surround the end of the journal 52 so as to stably support the journal 52. More particularly, the recesses 358 and 458 of the first and second bearings 300 and 400 substantially have a hemispherical shape. Thereby, the first and second bearings 300 and 400 support the shaft 50 in the axial direction of the shaft 50 as well as in the radial direction of the shaft 50. As a result, the above bearing structure stably supports the drum 30 as well as the shaft 50.

Preferably, the journal 52 of the shaft 50 has a cutting portion 58. A designated space 370, as shown in FIG. 12, is formed between the cutting portion 58 of journal 52 and the first and second bearings 300 and 400, and the grease is stored in the space 370. Thereby, the grease is sufficiently supplied to the first and second bearings 300 and 400 providing effective lubrication. Preferably, at least one oil groove (not shown) is formed in the outer circumferential surface of the journal 52. The oil groove always stores a designated amount of the grease, and supplies the grease to the space 370 between the journal 52 and the first and second bearings 300 and 400 providing effective lubrication.

The journal 52 is made of a metal, and is formed on

the body 54 by injection molding. Since the journal 52 is made of a metal, the abrasion of the journal 52 is drastically reduced. The first and second bearings 300 and 400 are made of plastic. Preferably, the first and second
5 bearings 300 and 400 are also formed by injection molding. Since the first and second bearings 300 and 400 can be finely manufactured and easily finished differing from a bearing made of a metal, surfaces of the first and second bearings 300 and 400 contacting the journal 52 have high
10 surface roughness. Thereby, the abrasion of the journal 52 is more reduced, and the generation of noise is reduced also.

Preferably, the surfaces of the first and second bearings 300 and 400 contacting the journal 52, i.e., the
15 surfaces of the recesses 358 and 458, are coated with a self lubricating material. A material containing Teflon and a material containing carbon are used as the self lubricating material. The above self lubricating material facilitates the effective lubrication of the journal 52,
20 and prevents the abrasion of the journal 52. The first and second bearings 150 and 250 of the first embodiment may employ the self lubricating material.

FIG. 18 is a sectional view of a second embodiment of the bearing structure of the drying machine in accordance

with the present invention, and FIG. 19 is an exploded perspective view of the second embodiment of the bearing structure. Hereinafter, the second embodiment of the bearing structure will be described in detail with
5 reference to FIGS. 18 and 19, as below.

As shown in FIGS. 18 and 19, the second embodiment of the bearing structure comprises a shaft 500 connected to the drum 30, a housing unit surrounding the shaft 500, and a bearing 800 installed in the first and second housings
10 600 and 700.

The shaft 500 comprises a flange 510 fixed to the rear surface of the drum 30, and a plurality of connection holes 501 are formed through the flange 510. The shaft 500 further comprises a journal 520 rotatably supported by the
15 bearing 800. The shaft 500 passes through the rear surface 3 of the cabinet 1, and the journal 520 is located at the outside of the cabinet 1. Accordingly, the housing unit and the bearing 800 are disposed at the outside of the cabinet 1 so as to support the journal 520.

20 The housing unit comprises first and second housings 600 and 700 substantially connected to each other. The first and second housings 600 and 700 respectively comprise seats 610 and 710 for mounting the bearing 800, and connection holes 601 and 701 for connecting the first and

second housings 600 and 700 to the rear surface 3 of the cabinet 1.

The bearing 800 is located in the seats 610 and 710 such that the bearing 800 can surround the journal 520, thereby rotatably supporting the shaft 500. The outer surface of the bearing 800 has a designated curvature, and then the seats 610 and 710 have a curvature corresponding to that of the bearing 800, as shown in FIGS. 18 and 19. Accordingly, the bearing 800 is stably installed in the seats 610 and 710 without using separate fixing members. Preferably, an oilless bearing is used as the bearing 800. The oilless bearing is made of a metal having many pores formed therein. Since the pores of the oilless bearing are filled with a lubricating oil, the oilless bearing does not require a separate lubricant.

In order to hold the bearing 800, rings 910 and 920 are installed on the shaft 500 at positions close to the bearing 800. A groove 570 is formed in the end of the shaft 500, and an E-ring 930 is installed in the groove 570. The E-ring 930 prevents the bearing 800 from being separated from the shaft 500, and alternately prevents the shaft 500 from being separated from the bearing 800. As shown in FIG. 18, in order to install the first and second housings 600 and 700 on the cabinet and to improve the

structural stability of the bearing structure, a bracket 33 is interposed between the cabinet and the first and second housings 600 and 700. The bracket 33 may be formed integrally with the cabinet.

5 It will be apparent to those skilled in the art that various modifications and variations can be made in the present invention without departing from the spirit or scope of the invention. Thus, it is intended that the present invention cover the modifications and variations of
10 this invention provided they come within the scope of the appended claims and their equivalents.

Industrial Applicability

Effect of the drying machine in accordance with the present invention will be described, as follows.

15 First, a bearing structure of the drying machine of the present invention is tightly sealed by grooves and ribs engaged with each other. Accordingly, since it is possible to prevent a grease from leaking from the bearing structure, the bearing structure is effectively lubricated, thereby
20 preventing the abrasion of a journal of a shaft and the generation of noise due to the abrasion of the journal.

Second, since bearings are made of plastic, the bearings have fine dimensions and are easily finished. Accordingly, the surfaces of the bearings contacting the

journal have high surface roughness, thereby efficiently preventing the abrasion of the journal.

Third, a bearing, which is coated with a self lubricating material, or an oilless bearing is used, 5 thereby providing effective lubrication so as to more efficiently prevent the abrasion of the journal.